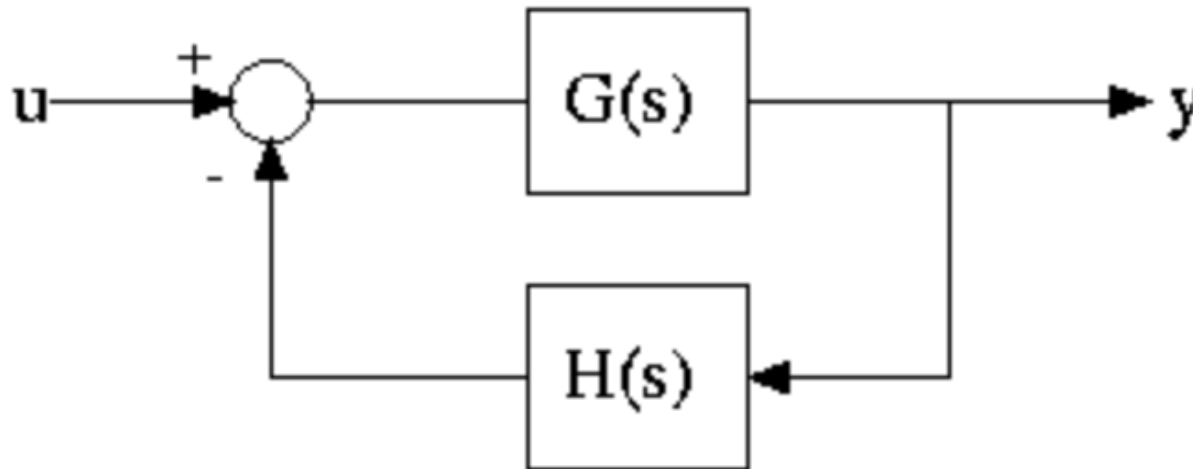


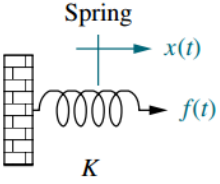
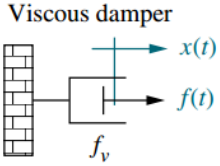
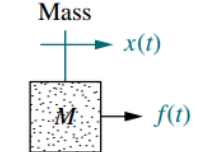
System Control

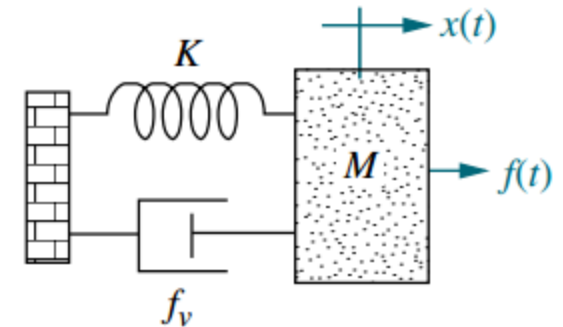
modeling in the frequency domain



Mechanical system (1)

- Translational mechanical system

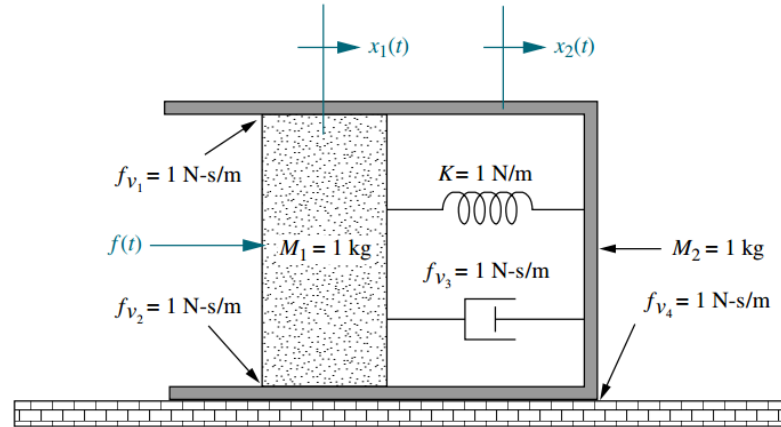
Component	Force-velocity	Force-displacement	Impedance $Z_M(s) = F(s)/X(s)$
 <p>Spring K</p>	$f(t) = K \int_0^t v(\tau) d\tau$	$f(t) = Kx(t)$	K
 <p>Viscous damper f_v</p>	$f(t) = f_v v(t)$	$f(t) = f_v \frac{dx(t)}{dt}$	$f_v s$
 <p>Mass M</p>	$f(t) = M \frac{dv(t)}{dt}$	$f(t) = M \frac{d^2 x(t)}{dt^2}$	Ms^2



$$(Ms^2 + f_v s + K)X(s) = F(s)$$

Note: The following set of symbols and units is used throughout this book: $f(t)$ = N (newtons), $x(t)$ = m (meters), $v(t)$ = m/s (meters/second), K = N/m (newtons/meter), f_v = N-s/m (newton-seconds/meter), M = kg (kilograms = newton-seconds²/meter).

Mechanical system (2)



• M_1 move M_2 hold

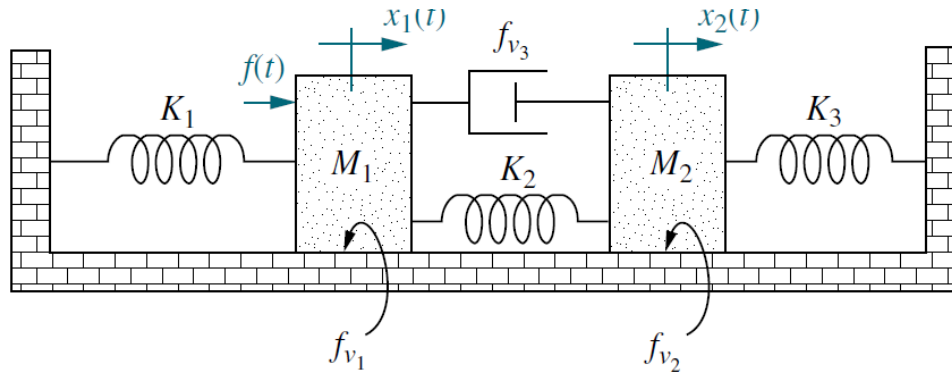
• M_1 hold M_2 move

• M_2 move M_1 hold

• M_2 hold M_1 move

Mechanical system (3)

Example Find the transfer function, $X_2(s)/F(s)$



• M_1 move M_2 hold

• M_1 hold M_2 move

• M_2 move M_1 hold

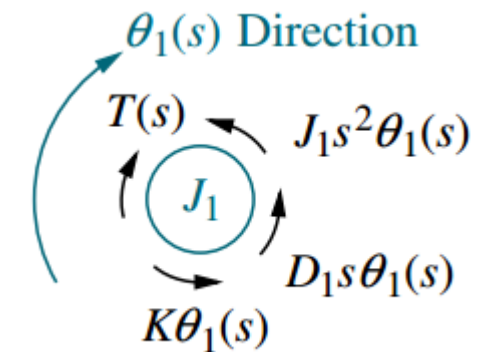
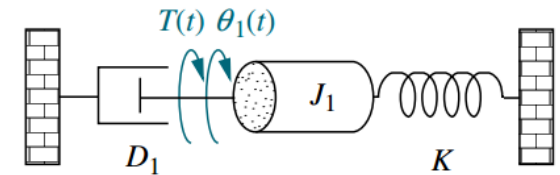
• M_2 hold M_1 move

Mechanical system (4)

• Rotational mechanical system

Component	Torque-angular velocity	Torque-angular displacement	Impedance $Z_M(s) = T(s)/\theta(s)$
<p>Spring K</p>	$T(t) = K \int_0^t \omega(\tau) d\tau$	$T(t) = K\theta(t)$	K
<p>Viscous damper D</p>	$T(t) = D\omega(t)$	$T(t) = D \frac{d\theta(t)}{dt}$	Ds
<p>Inertia J</p>	$T(t) = J \frac{d\omega(t)}{dt}$	$T(t) = J \frac{d^2\theta(t)}{dt^2}$	Js^2

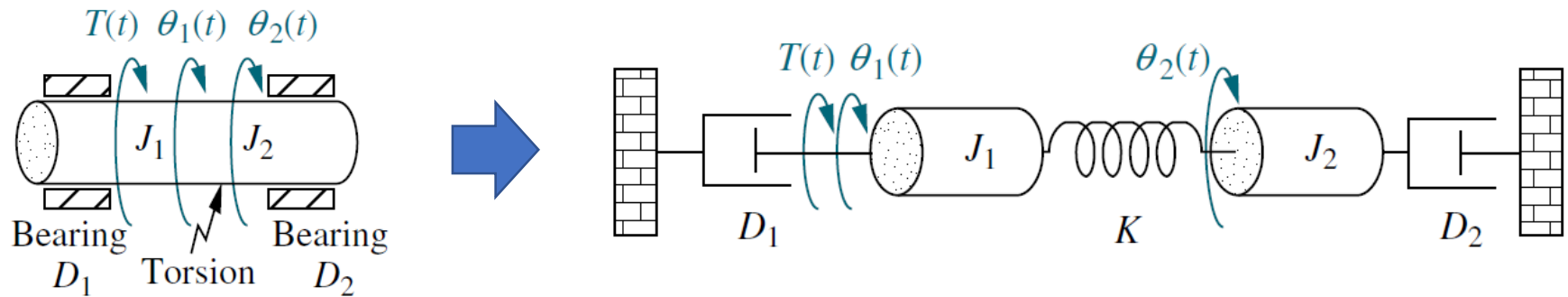
Note: The following set of symbols and units is used throughout this book: $T(t)$ – N-m (newton-meters), $\theta(t)$ – rad(radians), $\omega(t)$ – rad/s(radians/second), K – N-m/rad(newton- meters/radian), D – N-m-s/rad (newton- meters-seconds/radian). J – kg-m² (kilograms-meters² – newton-meters-seconds²/radian).



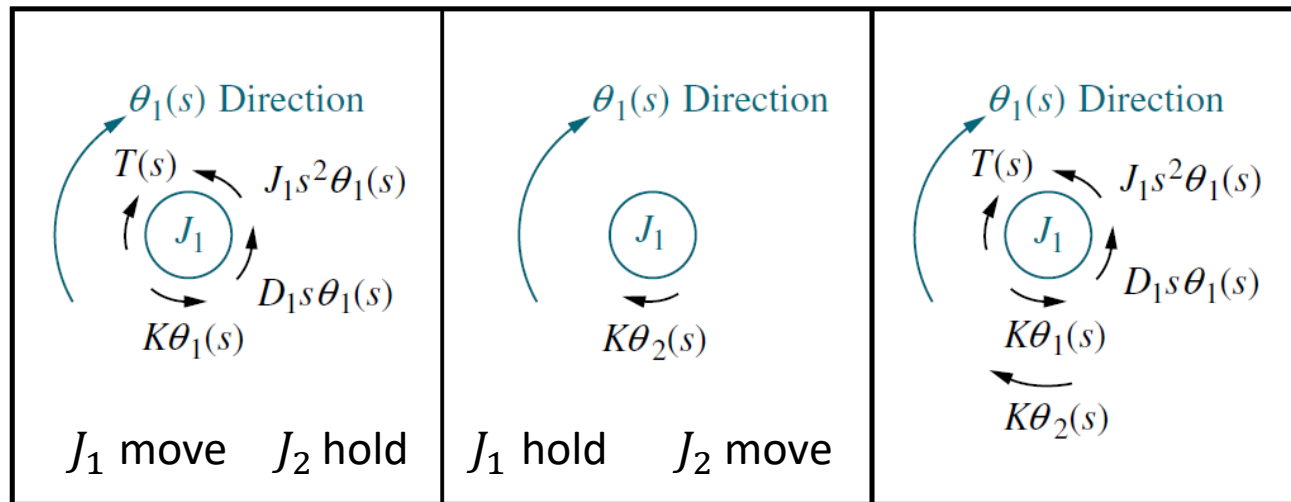
$$(J_1s^2 + D_1s + K)\theta_1(s) = T(s)$$

Mechanical system (5)

Example Find $G(s) = \theta_2(s)/T(s)$

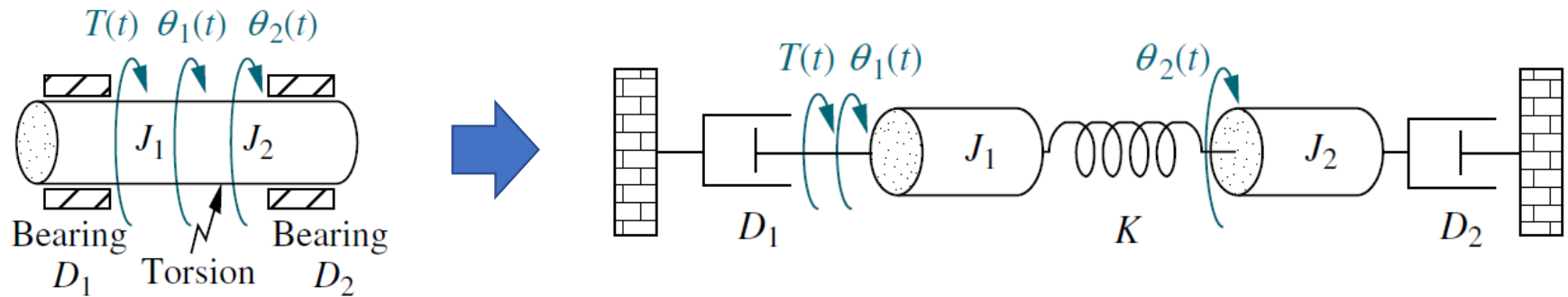


J_1

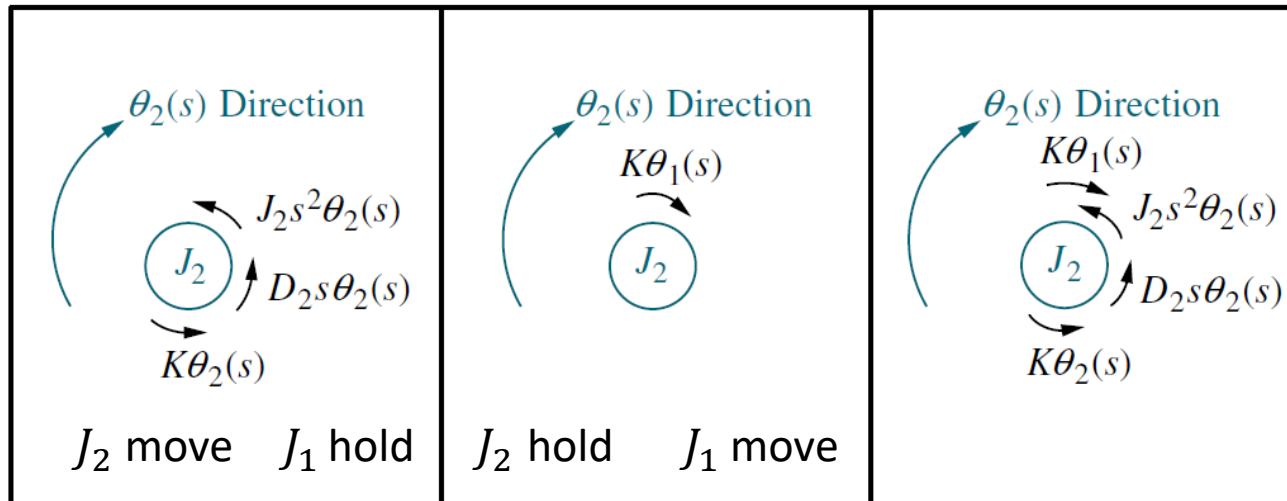


Mechanical system (6)

Example Find $G(s) = \theta_2(s)/T(s)$

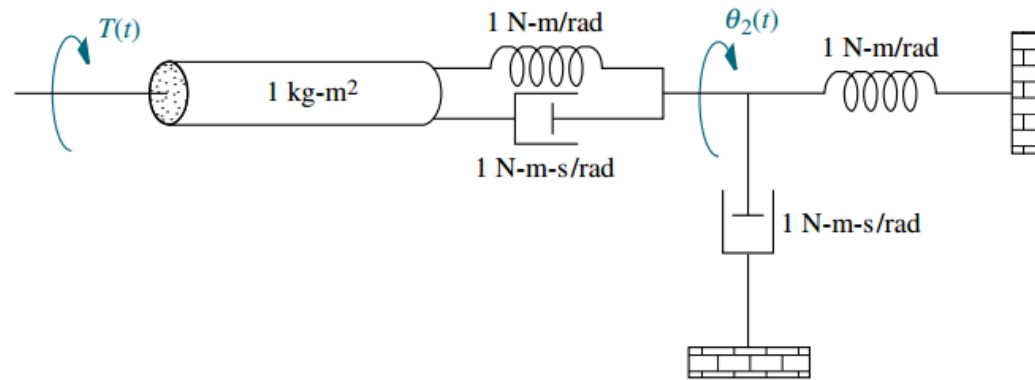


J_2



Mechanical system (7)

Find $G(s) = \theta_2(s)/T(s)$



• J_1 move J_2 hold

• J_1 hold J_2 move

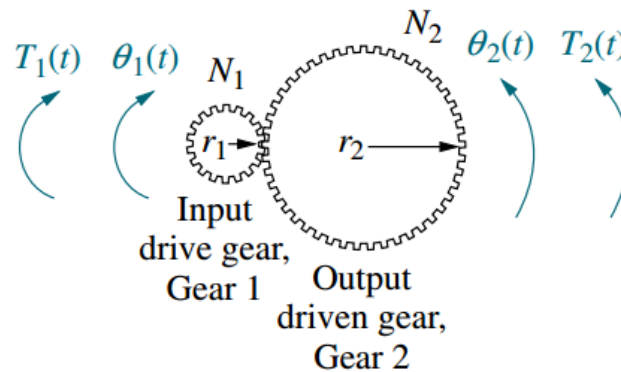
• J_2 move J_1 hold

• J_2 hold J_1 move

Mechanical system (8)

- Speed

$$r_1\theta_1 = r_2\theta_2$$



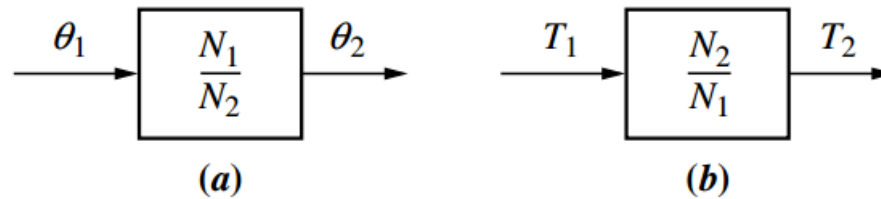
$$\frac{\theta_2}{\theta_1} = \frac{r_1}{r_2} = \frac{N_1}{N_2}$$

- Work

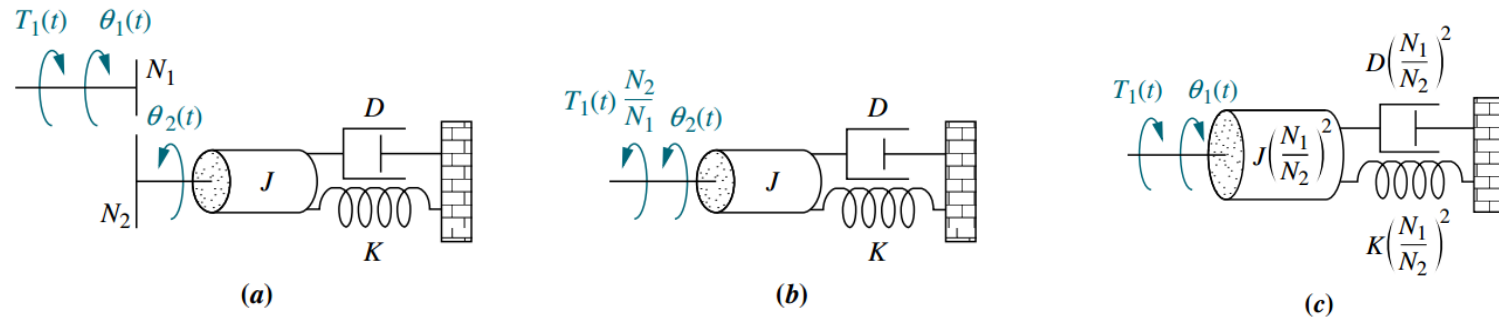
$$T_1\theta_1 = T_2\theta_2$$

cf.) $W = Fx$

$$\frac{T_2}{T_1} = \frac{\theta_1}{\theta_2} = \frac{N_2}{N_1}$$



Mechanical system (9)



Complex

Simple

$$(Js^2 + Ds + K)\theta_2(s) = T_1(s) \frac{N_2}{N_1}$$

$$\left[J \left(\frac{N_1}{N_2} \right)^2 s^2 + D \left(\frac{N_1}{N_2} \right)^2 s + K \left(\frac{N_1}{N_2} \right)^2 \right] \theta_1(s) = T_1(s)$$

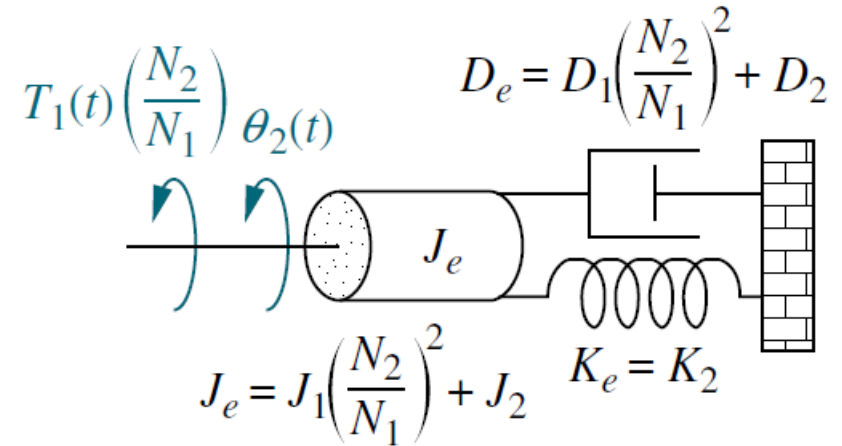
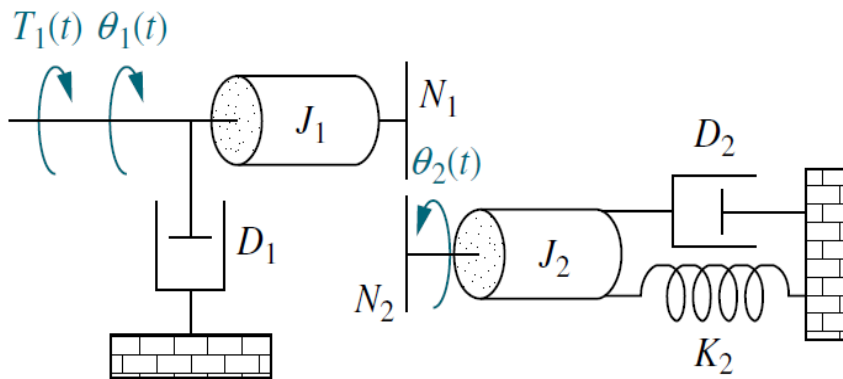


$$(Js^2 + Ds + K) \frac{N_1}{N_2} \theta_1(s) = T_1(s) \frac{N_2}{N_1}$$

$$\left(\frac{\text{Number of teeth of gear on destination shaft}}{\text{Number of teeth of gear on source shaft}} \right)^2$$

Mechanical system (10)

Find $G(s) = \theta_2(s)/T_1(s)$



$$(J_e s^2 + D_e s + K_e) \theta_2(s) = T_1(s) \frac{N_2}{N_1}$$

$$J_e = J_1 \left(\frac{N_2}{N_1}\right)^2 + J_2; \quad D_e = D_1 \left(\frac{N_2}{N_1}\right)^2 + D_2; \quad K_e = K_2$$

$$G(s) = \frac{\theta_2(s)}{T_1(s)} = \frac{N_2/N_1}{J_e s^2 + D_e s + K_e}$$

Electrical vs Mechanical

Same principle !!

$$\begin{bmatrix} \text{Sum of impedances connected to the motion at } \theta_1 \end{bmatrix} \theta_1(s) - \begin{bmatrix} \text{Sum of impedances between } \theta_1 \text{ and } \theta_2 \end{bmatrix} \theta_2(s) = \begin{bmatrix} \text{Sum of applied torques at } \theta_1 \end{bmatrix} \quad (2.129a)$$

$$- \begin{bmatrix} \text{Sum of impedances between } \theta_1 \text{ and } \theta_2 \end{bmatrix} \theta_1(s) + \begin{bmatrix} \text{Sum of impedances connected to the motion at } \theta_2 \end{bmatrix} \theta_2(s) = \begin{bmatrix} \text{Sum of applied torques at } \theta_2 \end{bmatrix} \quad (2.129b)$$

$$\begin{bmatrix} \text{Sum of impedances connected to the motion at } x_1 \end{bmatrix} X_1(s) - \begin{bmatrix} \text{Sum of impedances between } x_1 \text{ and } x_2 \end{bmatrix} X_2(s) = \begin{bmatrix} \text{Sum of applied forces at } x_1 \end{bmatrix} \quad (2.120a)$$

$$- \begin{bmatrix} \text{Sum of impedances between } x_1 \text{ and } x_2 \end{bmatrix} X_1(s) + \begin{bmatrix} \text{Sum of impedances connected to the motion at } x_2 \end{bmatrix} X_2(s) = \begin{bmatrix} \text{Sum of applied forces at } x_2 \end{bmatrix} \quad (2.120b)$$

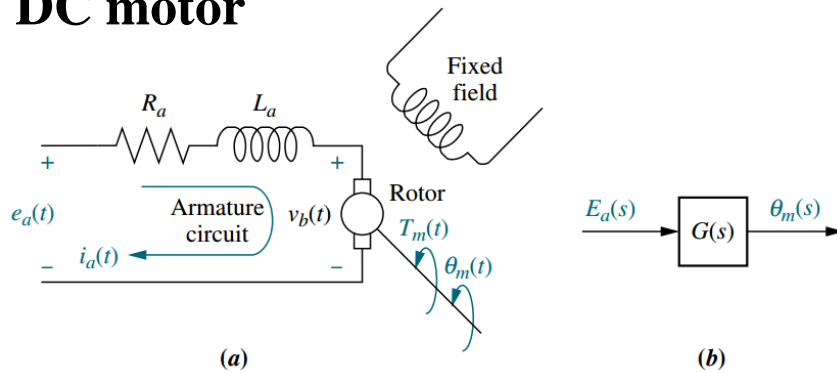


$$\begin{bmatrix} \text{Sum of impedances around Mesh 1} \end{bmatrix} I_1(s) - \begin{bmatrix} \text{Sum of impedances common to the two meshes} \end{bmatrix} I_2(s) = \begin{bmatrix} \text{Sum of applied voltages around Mesh 1} \end{bmatrix} \quad (2.83a)$$

$$- \begin{bmatrix} \text{Sum of impedances common to the two meshes} \end{bmatrix} I_1(s) + \begin{bmatrix} \text{Sum of impedances around Mesh 2} \end{bmatrix} I_2(s) = \begin{bmatrix} \text{Sum of applied voltages around Mesh 2} \end{bmatrix} \quad (2.83b)$$

Electromechanical system (1)

• DC motor



$$\frac{\theta_m(s)}{E_a(s)} = \frac{K_t / (R_a J_m)}{s \left[s + \frac{1}{J_m} \left(D_m + \frac{K_t K_b}{R_a} \right) \right]}$$

Electrical

$$R_a I_a(s) + L_a s I_a(s) + V_b(s) = E_a(s)$$



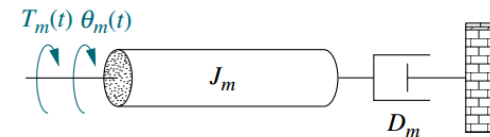
$$v_b(t) = K_b \frac{d\theta_m(t)}{dt}$$

$$T_m(s) = K_t I_a(s)$$

$$\frac{(R_a + L_a s) T_m(s)}{K_t} + K_b s \theta_m(s) = E_a(s)$$

Mechanical

$$T_m(s) = (J_m s^2 + D_m s) \theta_m(s)$$

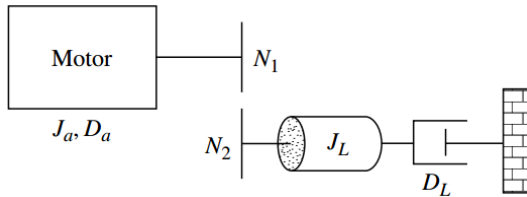


Electromechanical system (2)

- How to evaluate constants of motor?

$$\frac{\theta_m(s)}{E_a(s)} = \frac{K_t / (R_a J_m)}{s \left[s + \frac{1}{J_m} \left(D_m + \frac{K_t K_b}{R_a} \right) \right]} \rightarrow \left(J_m, D_m, \frac{K_t}{R_a}, K_b \right)$$

- Mechanical



$$J_m = J_a + J_L \left(\frac{N_1}{N_2} \right)^2 ; D_m = D_a + D_L \left(\frac{N_1}{N_2} \right)^2$$

- Electrical

$$L_a = 0, \quad \frac{R_a}{K_t} T_m(s) + K_b s \theta_m(s) = E_a(s)$$

$$T_m = -\frac{K_b K_t}{R_a} \omega_m + \frac{K_t}{R_a} e_a$$

$$T_{\text{stall}} = \frac{K_t}{R_a} e_a$$

$$\omega_{\text{no-load}} = \frac{e_a}{K_b}$$

$$\frac{K_t}{R_a} = \frac{T_{\text{stall}}}{e_a}$$

$$K_b = \frac{e_a}{\omega_{\text{no-load}}}$$

